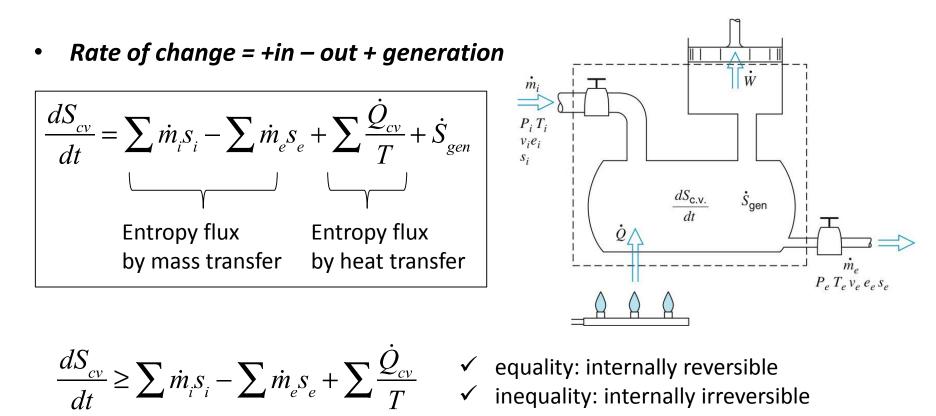
Chapter 7

Entropy Equation for a Control Volume

The Second Law for a Control Volume



* When the heat transfer is distributed over the control surface,

$$\sum \frac{\dot{Q}_{cv}}{T} = \int \frac{d\dot{Q}}{T} = \int_{surface} \frac{\left(\dot{Q} / A\right)_{local}}{T} dA$$

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The Steady-State Process and the Transient Process

• For a steady process,

$$\frac{dS_{cv}}{dt} = 0 \quad \Rightarrow \quad \sum \dot{m}_e s_e - \sum \dot{m}_i s_i = \sum_{cs} \frac{Q_{cv}}{T} + \dot{S}_{gen}$$

- For a steady process with a single inlet and a single outlet,

$$\dot{m}(s_e - s_i) = \sum_{cs} \frac{\dot{Q}_{cv}}{T} + \dot{S}_{gen}$$

$$s_e = s_i + \sum_{cs} \frac{q_{cv}}{T} + s_{gen} \xrightarrow{+ adiabatic} s_e = s_i + s_{gen} \xrightarrow{+ reversible} s_e = s_i$$

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• For a transient process,

$$\frac{d}{dt}\left(ms\right)_{cv} = \sum \dot{m}_{i}s_{i} - \sum \dot{m}_{e}s_{e} + \sum \frac{\dot{Q}_{cv}}{T} + \dot{S}_{gen}$$

- From integration with respect to time *t*,

$$\left(m_{2}s_{2}-m_{1}s_{1}\right)_{cv} = \sum m_{i}s_{i} - \sum m_{e}s_{e} + \int_{0}^{t} \sum \frac{Q_{cv}}{T}dt + {}_{1}S_{2gen}$$

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Besides, when the temperature is *uniform* throughout the control volume at any moment,

$$\int_{0}^{t} \sum \frac{\dot{Q}_{cv}}{T} dt = \int_{0}^{t} \frac{1}{T} \sum \dot{Q}_{cv} dt = \int_{0}^{t} \frac{\dot{Q}_{cv}}{T} dt$$
$$\left(m_{2}s_{2} - m_{1}s_{1}\right)_{cv} = \sum m_{i}s_{i} - \sum m_{e}s_{e} + \int_{0}^{t} \frac{\dot{Q}_{cv}}{T} dt + {}_{1}S_{2gen}$$

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The Steady-State Single-Flow Process

Now we derive the expression for the work in a *steady-state, single flow process*.

Energy Eq.:
$$q + h_i + \frac{1}{2}V_i^2 + gZ_i = h_e + \frac{1}{2}V_e^2 + gZ_e + w$$

Entropy Eq.: $s_e = s_i + \int \frac{\delta q}{T} + s_{gen}$

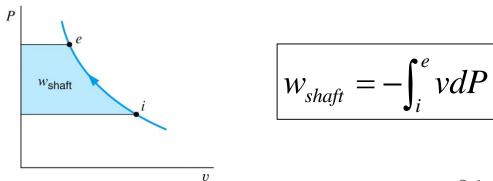
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$$w_{shaft} = -\int_{i}^{e} v dP + \frac{1}{2} (V_{i}^{2} - V_{e}^{2}) + g(Z_{i} - Z_{e}) - \int_{i}^{e} T \delta s_{gen}$$

- The above expression is useful to account for the work of turbines, compressors, and pumps.
- ✓ The maximum work is produced when the entropy generation is zero.
- ✓ For a reversible process, work can be extracted from decrease in pressure , kinetic energy or potential energy.
- \checkmark A large amount of work is involved when the specific volume is large.
- In a real fluid flow without shaft work or change in the kinetic energy and the potential energy, pressure decreases due to friction.
- When the process is <u>reversible and steady without change in ke and pe</u>, the above expression reduces to



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• When the process is *reversible* and the fluid is *incompressible*,

$$w_{shaft} = -v(P_e - P_i) + \frac{1}{2}(V_i^2 - V_e^2) + g(Z_i - Z_e)$$

- For zero work,

$$vP_i + \frac{1}{2}V_i^2 + gZ_i = vP_e + \frac{1}{2}V_e^2 + gZ_e$$
 (Bernoulli equation)

* For a steady, reversible polytropic process for an ideal gas with no change in kinetic and potential energies, we have the following relation.

$$w = -\int_{i}^{e} v dP \text{ and } Pv^{n} = \text{constant} = C^{n}$$

$$w = -\int_{i}^{e} v dP = -C \int_{i}^{e} \frac{dP}{P^{1/n}} = -\frac{n}{n-1} (p_{e}v_{e} - p_{i}v_{i}) = -\frac{nR}{n-1} (T_{e} - T_{i}) \quad (n \neq 1)$$

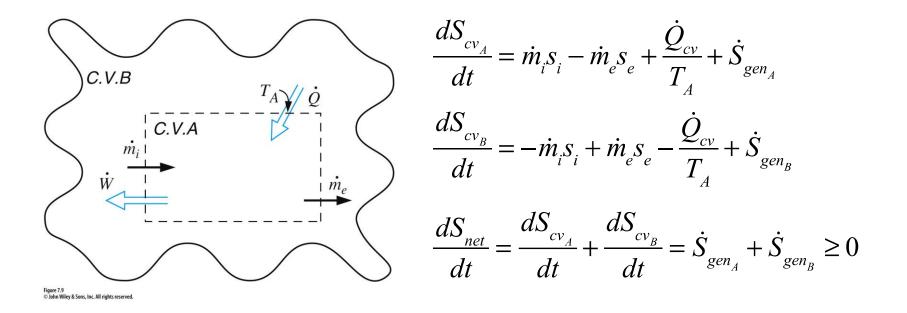
or

$$w = -\int_{i}^{e} v dP = -\text{constant} \int_{i}^{e} \frac{dP}{P} = -P_{i}v_{i} \ln \frac{P_{e}}{P_{i}} \qquad (n=1)$$

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Principle of the Increase of Entropy



 In typical mechanical devices with multiple valves used to control the fluid flow, they introduce irreversible throttling processes besides having an irreversible mixing and possible heat transfer over finite temperature differences.



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